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瑞利波作用下考虑竖向荷载影响的单桩在饱和软土中的水平动力分析

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摘要: 桩在竖向荷载和瑞利波共同作用下产生水平振动, 竖向荷载会因二阶效应导致水平位移增大。为研究桩在饱和软土地基中的水平动力响应, 建立瑞利波作用下单桩动力响应的计算模型。基于 Biot 理论计算均匀自由场中饱和软土地基的水平动力响应。利用边界条件求得土体阻力封闭解。基于 Timoshenko 梁理论建立桩基动力微分方程, 得到桩的水平位移、弯矩和转角的解析解。通过数值算例验证模型正确性, 分析竖向荷载、无量纲频率对桩水平振动的影响。研究发现竖向荷载和桩长对水平振动影响较大。

关键词: 饱和软土; 柔性约束; 水平振动; 瑞利波; Timoshenko 梁

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Horizontal dynamic analysis of a single pile in saturated soft soils under Rayleigh wave action considering effects of vertical loads

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Abstract: The pile generates horizontal vibration under the combined action of vertical loads and Rayleigh waves, and the vertical loads will increase the horizontal displacement due to the second-order effects. In order to study the horizontal dynamic response of piles in saturated soft soil foundation, a model for calculating the dynamic response of a single pile under the action of Rayleigh waves is established. Based on the Biot theory, the horizontal dynamic response of saturated soft soil foundation in a uniform free field is calculated. The boundary conditions are used to obtain the closure solution of soil resistance. Based on the Timoshenko beam theory, the dynamic differential equation for the pile foundation is established, and the analytical solutions of the horizontal displacement, bending moment and rotation angle of the pile are obtained. Numerical examples are used to verify the correctness of the model, and the influences of the vertical load and dimensionless frequency on the horizontal vibration of the pile are analyzed. It is found that the vertical load and pile length have a great influence on the horizontal vibration.

Key words: saturated soft soil; flexible constraint; horizontal vibration; Rayleigh wave; Timoshenko beam

0 引言

瑞利波传播过程中主要引起水平方向振动, 这种振动与建筑物的自然频率相近时, 会引起共振, 导致更大的破坏。为进一步明确瑞利波传播特性和衰减规律, 大量学者对其进行理论分析^[1]、数值模拟^[2]和试验研究^[3]。瑞利波在介质中的传播受多种因素影响, 主要包括介质的弹性参数、密度、重力以及分层结构等^[4]。

桩基在瑞利波作用下的动力响应的研究成为抗震过程中的重要组成部分。Yang 等^[5]在考虑桩顶柔性约

束的情况下, 研究了非饱和土-桩体系在瑞利波作用下的动力响应。此外 Cai 等^[6]采用刚性排桩作为隔离瑞利波的方法, 考虑了软土地质所产生的影响。

通过以上讨论发现: 考虑竖向荷载影响的瑞利波作用下桩水平动力响应分析比较复杂。且对于软土地基中桩在瑞利波作用下的动力响应的研究也相对较少。因此, 本文建立考虑竖向荷载对瑞利波作用下饱和软土地基中桩的水平动力响应影响的计算模型。研

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究结果为瑞利波作用下桩基动力分析和设计提供理论指导。

1 计算模型

1.1 问题描述

在瑞利波的作用下饱和土-桩体系的数学模型如图 1 所示, 其中桩长为 L , 半径为 r_a , 截面积为 A_p , 密度为 ρ_p , 杨氏模量为 E_p , 剪切模量为 G_p 。桩体模型简化为桩端为固定边界条件的 Timoshenko 梁, 桩顶视为质量为 M_0 的刚性块体。桩周饱和土视为线弹性材料, 泊松比、阻尼比和剪切模量分别取为 ν_0 , ε_0 , s 和 G_s 。其中瑞利波的作用方式是以波的形式穿过土体传播到桩体上。

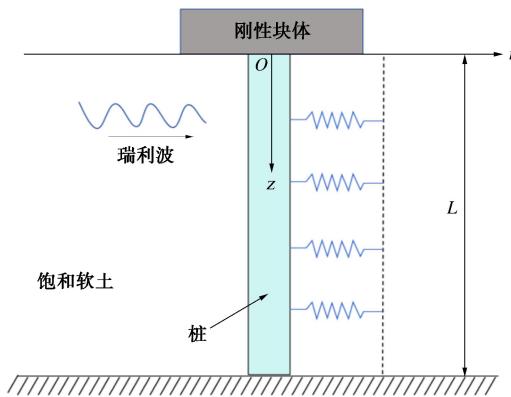


图 1 瑞利波作用下桩在饱和土中的计算模型

Fig. 1 Computational model for pile in saturated soils under Rayleigh waves

1.2 基本方程

根据 Biot's 理论, 运动方程可以表示为

$$\sigma_{ij,j} = \rho_s \ddot{u}_i + \rho_f \ddot{w}_i \quad , \quad (1)$$

$$-p_{f,i} = \rho_f \ddot{u}_i + m_1 \ddot{w}_i + r_1 \dot{w}_i \quad . \quad (2)$$

式中: u_i 为土体位移; w_i 为流体位移; λ , μ 为拉梅常数; $m_1 = \rho_f/n$ 为孔隙中流体密度与孔隙率的比值; $r_1 = \rho_f g/k_d$, k_d 为土壤达西渗透系数, p_f 为孔隙中流体压力; α 和 M 分别为两相材料的 Biot's 参数。

基于 Timoshenko 理论, 建立桩段控制微分方程^[7]:

$$A_p G_p k_0 \left(\frac{\partial^2 u_p}{\partial z^2} - \frac{\partial \theta_p}{\partial z} \right) + \rho_p A_p \frac{\partial^2 u_p}{\partial t^2} + (k_h + i c_h)(u_p - u_r \cos \theta) = 0 \quad , \quad (3)$$

$$A_p G_p k_0 \left(\frac{\partial u_p}{\partial z} - \theta_p \right) - \rho_p I_p \frac{\partial^2 u_p}{\partial t^2} - E_p I_p \frac{\partial^2 \theta_p}{\partial z^2} + A_p P \frac{\partial u_p}{\partial z} = 0 \quad . \quad (4)$$

式中: $u_p(z, t) = \bar{u}_p(z) e^{i \omega t}$, $\theta_p(z, t) = \bar{\theta}_p(z) e^{i \omega t}$ 分别为桩的水平位移和转角; k_0 为 Timoshenko 梁理论中的修正

剪切因子; P 为由上部结构质量引起的竖向荷载。

2 方程的求解

由式 (2) 得:

$$w_r = \frac{\omega^2 \rho_f u_r}{g} - \frac{1}{g} \frac{\partial p_f}{\partial r}, \quad w_\theta = \frac{\omega^2 \rho_f u_\theta}{g} - \frac{1}{g} \frac{\partial p_f}{\partial \theta} \quad . \quad (5)$$

式中: $\eta_a^2 = 2/(1-\nu)$, $\vartheta = i\omega \rho_f g/k_d - m_1 \omega^2$ 。

将式 (5) 代入式 (1), (2) 结合本构方程得

$$\eta_b^2 \frac{\partial}{\partial r} \left[\frac{1}{r} \left(\frac{\partial (r u_r)}{\partial r} + \frac{\partial u_\theta}{\partial \theta} \right) \right] - \frac{1}{r^2} \frac{\partial}{\partial \theta} \left[\frac{\partial (r u_\theta)}{\partial r} - \frac{\partial u_r}{\partial \theta} \right] + \frac{\partial^2 u_r}{\partial z^2} + c_1 \frac{\rho_s \omega^2}{G_s} u_r - c_2 \frac{\partial p_f}{\partial r} = 0 \quad , \quad (6)$$

$$\frac{\partial}{\partial r} \left[\frac{1}{r} \left(\frac{\partial (r u_\theta)}{\partial r} - \frac{\partial u_r}{\partial \theta} \right) \right] + \eta_b^2 \frac{1}{r^2} \frac{\partial}{\partial \theta} \left[\frac{\partial (r u_r)}{\partial r} + \frac{\partial u_\theta}{\partial \theta} \right] + \frac{\partial^2 u_\theta}{\partial z^2} + c_1 \frac{\rho_s \omega^2}{G_s} u_\theta - c_2 \frac{\partial p_f}{\partial r \partial \theta} = 0 \quad . \quad (7)$$

式中: $c_1 = 1 + \frac{\omega^2 \rho_f^2}{\rho_s g}$; $c_2 = \frac{\alpha}{2\mu} \frac{1-2\nu}{1-\nu} + \frac{\omega^2 \rho_f}{\mu_s g}$; $\eta_b^2 = \frac{2-\nu}{1-\nu}$

忽略流体的垂直位移:

$$\left(\frac{\omega^2 \rho_f}{g} - \alpha \frac{1-2\nu_0}{1-\nu_0} \right) \left(\frac{\partial u_r}{\partial r} + \frac{u_r}{r} + \frac{1}{r} \frac{\partial u_\theta}{\partial \theta} \right) = \frac{1}{g} \nabla^2 p_f + \left(\frac{\alpha^2}{\lambda} \frac{\nu_0}{1-\nu_0} + \frac{1}{M} \right) p_f \quad . \quad (8)$$

水平位移和切向位移用两个势函数 Φ 和 Ψ 表示为

$$u_r = \frac{\partial \phi}{\partial r} + \frac{1}{r} \frac{\partial \psi}{\partial \theta}, \quad u_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} - \frac{\partial \psi}{\partial r} \quad . \quad (9)$$

将式 (9) 代入式 (6) ~ (8), 联立得

$$\eta_b^2 \nabla^2 \phi + \frac{\partial^2 \phi}{\partial z^2} + c_1 \frac{\rho_s}{G_s} \omega^2 \phi = c_2 p_f \quad , \quad (10)$$

$$\nabla^2 \psi + \frac{\partial^2 \psi}{\partial z^2} + c_1 \frac{\rho_s}{G_s} \omega^2 \psi = 0 \quad , \quad (11)$$

$$\nabla^2 p_f + \left(\frac{9\alpha^2}{\lambda} \frac{\nu_0}{1-\nu_0} + \frac{g}{M} \right) p_f = \left(\omega^2 \rho_f - \alpha g \frac{1-2\nu_0}{1-\nu_0} \right) \nabla^2 \phi \quad . \quad (12)$$

采用分离变量法, 设 $\phi(r, \theta, z) = \phi_1(r, \theta)Z(z)$, 假设 $\phi(r, \theta, z) = \phi_1(r, \theta)Z(z)$, $\frac{d^2 Z}{dz^2} + a^2 Z = 0$, 将 $Z(z)$ 表示为

$$Z(z) = B_0 \cos(az) + B_1 \sin(az) \quad . \quad (13)$$

联立式 (10) ~ (12) 并将式 (13) 代入得

$$\nabla^2 \psi_1 - \left[a^2 - c_1 \frac{\rho_s}{G_s} \omega^2 \right] \psi_1 = 0 \quad . \quad (14)$$

$$(\nabla^2 - \zeta_{11}^2)(\nabla^2 - \zeta_{12}^2) \phi_1 = 0 \quad . \quad (15)$$

这里设:

$$d_{11} = \frac{c_1}{\eta_b^2} \frac{\rho_s}{G_s} \omega^2 + \frac{\vartheta}{M} - \frac{a^2}{\eta_b^2} + \frac{c_2 \alpha \vartheta}{\eta_b^2} \frac{1-2v_0}{1-v_0} - \frac{c_2 \omega^2 \rho_f}{\eta_b^2} + \frac{9\alpha^2}{\lambda} \frac{v_0}{1-v_0}, \quad d_{12} = \frac{\vartheta}{\eta_b^2} \left[c_1 \frac{\rho_s}{G_s} \omega^2 \left(\frac{\alpha^2}{\lambda} \frac{v_0}{1-v_0} + \frac{1}{M} \right) - \frac{a^2 \alpha^2}{\lambda} \frac{v_0}{1-v_0} - \frac{a^2}{M} \right], \quad \zeta_{11}^2 = \frac{-d_{11} + \sqrt{d_{11}^2 - 4d_{12}}}{2}, \quad \zeta_{12}^2 = \frac{-d_{11} - \sqrt{d_{11}^2 - 4d_{12}}}{2}, \quad v_s = \sqrt{\frac{G_s}{\rho_s}}, \quad \text{从而得到}$$

$$(\nabla^2 + k_{a1}^2)(\nabla^2 + k_{a2}^2)\phi_1 = 0, \quad (16)$$

$$(\nabla^2 + k_{b1}^2)\psi_1 = 0. \quad (17)$$

通过算子分解理论和分离变量法得：

$$\phi_1 = A_{s1} \exp(-s_1 r \sin \theta - ik_R r \cos \theta) + A_{s2} \exp(-s_2 r \sin \theta - ik_R r \cos \theta), \quad (18)$$

$$\psi_1 = B_s \exp(-\gamma r \sin \theta - ik_R r \cos \theta). \quad (19)$$

式中： A_{s1}, A_{s2}, B_s 分别为与边界条件有关的待定常数； $k_R, V_R = \omega/k_R$ 分别为瑞利波的复波速和相速度； $s_i = \sqrt{k_R^2 - k_{ai}^2}$ ($i=1,2$) 和 $\gamma = \sqrt{k_R^2 - k_{b1}^2}$ ，分别为压缩波和剪切波对应的衰减指数； $k_{a1}^2 = -\zeta_{11}^2, k_{a2}^2 = -\zeta_{12}^2, k_{b1}^2 = -a^2 + \kappa_1 \frac{\rho_s}{G_s} \omega^2$ 。

因此由式 (13)，结合边界条件即 $\tau_{rz}|_{z=0} = 0, u_r|_{z=L} = u_\theta|_{z=L} = 0$ 可知

$$B_0 = B_1 = 0 \text{ 且 } \cos(aL) = 0, \quad \left. \begin{aligned} a_m &= \frac{(2m-1)\pi}{2L}, m=1,2,3 \end{aligned} \right\} \quad (20)$$

$$\varphi = [A_{s1} \exp(-s_1 r \sin \theta - ik_R r \cos \theta) + A_{s2} \exp(-s_2 r \sin \theta - ik_R r \cos \theta)] \sin(a_m z), \quad (21)$$

$$\psi = B_s \exp(-\gamma r \sin \theta - ik_R r \cos \theta) \sin(a_m z). \quad (22)$$

将式 (21) 代入式 (10) 得

$$p_f = \left[-\frac{a_m^2}{c_2} + \frac{c_1}{c_2} \left(\frac{\omega}{V_s} \right)^2 \right] \left[A_{s1} \exp\left(-s_1 r \sin \theta - ik_R r \cos \theta\right) + A_{s2} \exp\left(-s_2 r \sin \theta - ik_R r \cos \theta\right) \right] \sin(a_m z) + \frac{\eta_b^2}{c_2} (s_1^2 - k_R^2) A_{s1} \exp(-s_1 r \sin \theta - ik_R r \cos \theta) \sin(a_m z) + \frac{\eta_b^2}{c_2} (s_2^2 - k_R^2) A_{s2} \exp(-s_2 r \sin \theta - ik_R r \cos \theta) \sin(a_m z). \quad (23)$$

通过式 (9) 结合本构方程及式 (21), (22) 并代入边界条件 $\tau_{rz}|_{z=0} = 0, u_\theta|_{z=L} = 0$ 可知

$$A_{s2} = d_1 A_{s1}, B_s = d_2 A_{s1}. \quad (24)$$

联立式 (9) 即可求得位移转角 u_r, u_θ 表达式。

因此自由场饱和土在瑞利波作用下位移由体积平均原理得

$$\bar{u}_r = (1-n)u_r + nw_r, \quad (25)$$

$$\bar{u}_\theta = (1-n)u_\theta + nw_\theta. \quad (26)$$

3 瑞利波作用下桩的动力响应

3.1 水平阻力

桩在瑞利波作用下的动力响应由桩周土体决定，采用动力 Winkler 模型来描述饱和土桩体系的水平动力响应，进而计算单位桩长上的水平阻力^[8-9]。因此设单位桩长上的水平阻力为

$$q_h = (k_h + i c_h) u_{\bar{p}}. \quad (27)$$

式中： k_h 和 c_h 分别为 Winkler 模型中弹簧刚度和阻尼系数。

3.2 考虑竖向荷载影响的桩的动力响应

将式 (25), (27) 代入式 (3), (4) 并省略因子 $e^{i\omega t}$ 得

$$\frac{d^4 \bar{u}_p}{dz^4} + W \frac{d^2 \bar{u}_p}{dz^2} + J \bar{u}_p = H_1 A_{s1} \sin(a_m z), \quad (28)$$

$$\frac{d^4 \bar{\theta}_p}{dz^4} + W \frac{d^2 \bar{\theta}_p}{dz^2} + J \bar{\theta}_p = H_2 A_{s1} \cos(a_m z). \quad (29)$$

其中， $W = \frac{\rho_p \omega^2}{E_p} + \frac{\rho_p \omega^2}{k_0 G_p} + \frac{A_p P}{E_p I_p} - \frac{(k_h + i c_h)}{k_0 A_p G_p}, \quad J = \frac{\rho_p^2 \omega^4}{k_0 G_p E_p} - \frac{\rho_p A_p \omega^2}{E_p I_p} - \left(\frac{\rho_p \omega^2}{k_0 A_p G_p E_p} - \frac{1}{E_p I_p} \right) (k_h + i c_h)$

方程 (28)、(29) 对应的解设为

$$\bar{u}_p = M_1 \cos(\lambda_1 z) + M_2 \sin(\lambda_1 z) + M_3 \operatorname{ch}(\lambda_2 z) + M_4 \operatorname{sh}(\lambda_2 z) + b_1 A_{s1} \sin(a_m z), \quad (30)$$

$$\bar{\theta}_p = M_1 \chi_1 \sin(\lambda_1 z) + M_2 \chi_2 \cos(\lambda_1 z) + M_3 \chi_3 \operatorname{sh}(\lambda_2 z) + M_4 \chi_4 \operatorname{ch}(\lambda_2 z) + b_2 A_{s1} \cos(a_m z), \quad (31)$$

式中： $\lambda_1 = \sqrt{\frac{W + \sqrt{W^2 - 4J}}{2}}, \lambda_2 = \sqrt{\frac{-W + \sqrt{W^2 - 4J}}{2}}$ ，
 $M_i (i=1, 2, 3, 4)$ 为与桩的边界条件有关的待定系数。

其中 $\chi_1 = -\chi_2 = -\frac{k_0 A_p G_p \lambda_1 + A_p P \lambda_1}{k_0 A_p G_p + \rho_p I_p \omega^2 + E_p I_p \lambda_1^2}, \quad \chi_3 =$

$$\chi_4 = \frac{k_0 A_p G_p \lambda_2 + A_p P \lambda_2}{k_0 A_p G_p + \rho_p I_p \omega^2 - E_p I_p \lambda_1^2}, \quad b_1 = \frac{H_1}{a_m^4 - W a_m^2 + J},$$

$$b_2 = \frac{H_2}{a_m^4 - W a_m^2 + J}.$$

沿桩身的弯矩和剪力可由弹性力学推导为

$$\bar{M}_p = E_p I_p [M_1 \chi_1 \lambda_1 \cos(\lambda_1 z) - M_2 \chi_2 \lambda_1 \sin(\lambda_1 z) + M_3 \chi_3 \lambda_2 \operatorname{ch}(\lambda_2 z) + M_4 \chi_4 \lambda_2 \operatorname{sh}(\lambda_2 z) - b_2 a_m A_{s1} \sin(a_m z)]. \quad (32)$$

$$\begin{aligned} \bar{Q}_p &= k_0 A_p G_p [-M_1(\lambda_1 + \chi_1) \sin(\lambda_1 z) + M_2(\lambda_1 - \chi_2) \cdot \\ &\cos(\lambda_1 z) + M_3(\lambda_2 - \chi_3) \sinh(\lambda_2 z) + M_4(\lambda_2 - \chi_4) \cosh(\lambda_2 z) + \\ &(b_1 a_m - b_2) A_{s1} \cos(a_m z)] \quad (33) \end{aligned}$$

桩顶柔性状态下, 桩端处于固定状态, 桩的边界条件为:

$$\left. \begin{aligned} \bar{M}_p(z) &= K_1 \bar{\theta}_c + K_2 [\bar{\theta}_p(0) - \bar{\theta}_c], \quad z=0, \\ \bar{Q}_p(z) - A_p P \bar{\theta}_p(z) + \omega^2 M_0 \bar{u}_p(z) &= 0, \quad z=0, \\ \bar{u}_p(z) = 0, \quad \bar{\theta}_p(z) = 0, &z=L \end{aligned} \right\} \quad (34)$$

将边界条件代入桩体水平位移、旋转角度沿桩身的弯矩和剪力表达式, 由此可以导出所有未确定的待定参数 M_i ($i=1,2,3,4$), A_{s1} 。

4 数值分析及讨论

通过数值算例验证, 分析竖向荷载及桩顶柔性约束下各参数对于桩身位移、转角和弯矩的影响。各相关参数属性: $\rho_s = 2.7 \times 10^3 \text{ kg/m}^3$, $\rho_f = 2.2 \times 10^3 \text{ kg/m}^3$, $E_a = 2 \times 10^9 \text{ Pa}$, $\nu_0 = 0.4$, $n = 0.4$, $\alpha = 0.9$, $k_d = 1 \times 10^{-8} \text{ m/s}$, $K_s = 3.6 \times 10^{10} \text{ Pa}$, $K_f = 2 \times 10^9 \text{ Pa}$, $G_s = 2.5 \times 10^6 \text{ Pa}$, $d = 1 \text{ m}$, $r_a = 0.5 \text{ m}$, $L = 20 \text{ m}$, $\rho_p = 2.5 \times 10^3 \text{ kg/m}^3$, $E_p = 2.5 \times 10^{10} \text{ Pa}$, $k_0 = 0.75$, $I_p = \pi/64$, $\nu_p = 0.2$, $A_p = 0.25\pi$, $M_0 = 1 \times 10^5 \text{ kg}$, $P = 10 \times 10^6 \text{ Pa}$ 。引入无量纲频率 $a_0 = \omega L_a / \nu_s$ 。

4.1 验证

为验证模型的准确性, 将 Makris^[10]解与模型进行比较。在相同条件下, 对本文的计算模型进行验证。图 2 将水平位移与 Makris^[10]解的结果进行比较。可以看出两种解法有较高的一致性。

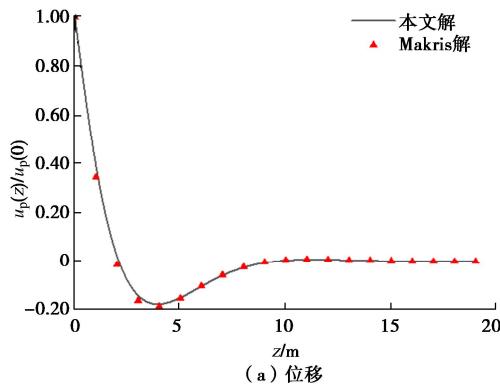


图 2 本文解与 Makris 解的比较

Fig. 2 Comparison between authors' and Makris' solutions

4.2 参数分析

图 3, 4 给出了瑞利波作用下, 竖向荷载对单桩的水平动力响应的影响, 其研究了桩长、竖向荷载大小和无量纲频率对桩基水平振动的影响。从图 3 可以看出, 随着竖向荷载的增加, 位移、转角和弯矩都在增大, 且增加的趋势也越来越大。随着深度的增加, 竖向荷载改变所引起的变化不再明显, 最终趋于稳定。

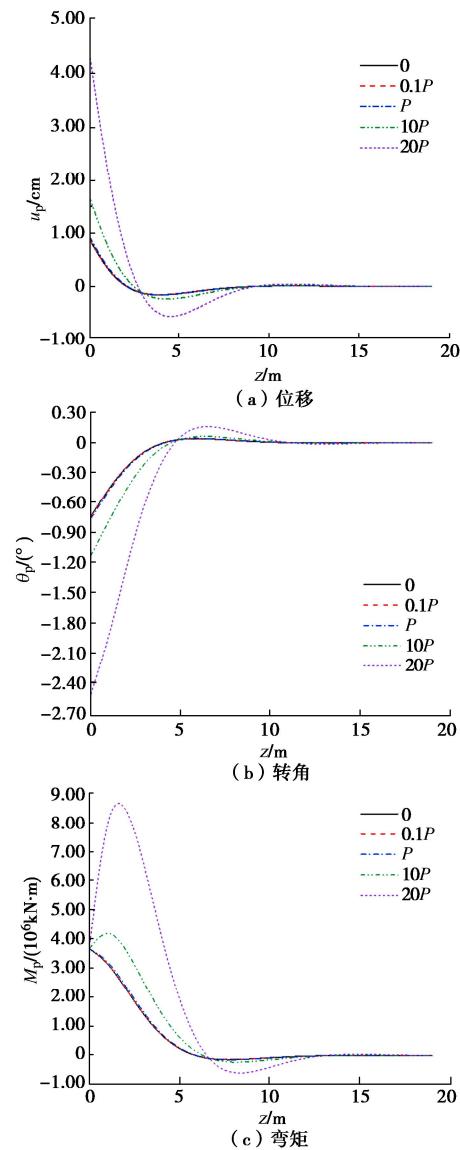
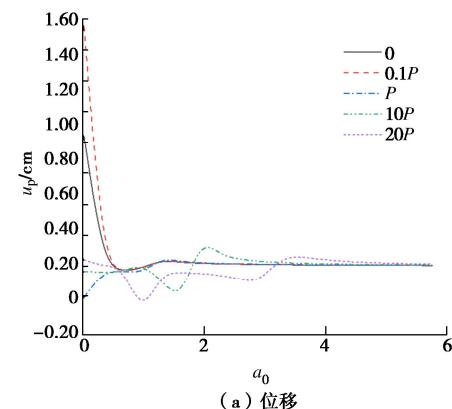


图 3 竖向荷载对单桩横向响应沿桩长变化的影响

Fig. 3 Influences of vertical load on lateral response of a single pile along pile length

图 4 显示了垂直载荷对位移、转角和弯矩的影响。当频率等于固有频率时, 发生共振。在共振区, 当竖向荷载较小时, 随竖向荷载的增大, 桩顶位移和桩端弯矩整体呈增大趋势。随无量纲频率的增大, 竖向荷载的改变引起水平位移和转角的变化将不再明显。



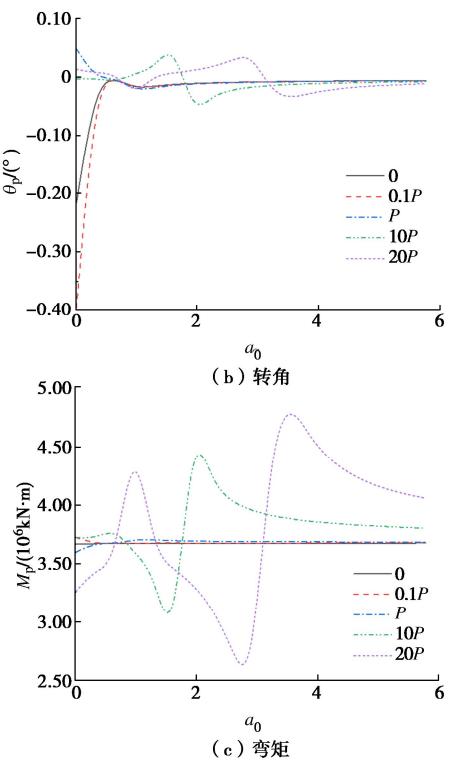


图 4 坚向荷载对单桩水平响应随无因次频率变化的影响
Fig. 4 Influences of vertical load on horizontal response of a single pile with dimensionless frequency

5 结 论

考虑坚向荷载对瑞利波作用下饱和软土中单桩结构水平动力响应的影响, 建立瑞利波作用下饱和软土中桩顶柔性约束下单桩水平动力响应的计算模型。通过数值计算结果, 得出以下 3 点结论。

(1) 随深度的增加, 位移和转角先减小后增大; 弯矩整体呈减小趋势; 最终接近桩端固结端时桩的水平动力响应趋于稳定。

(2) 随无量纲频率的增加, 位移、转角和弯矩在共振区发生共振后最终趋于稳定。

(3) 随坚向荷载的增加, 位移、转角和弯矩都在增大。但是这种改变所引起的变化随着深度的增加不再明显, 最终趋于稳定。

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